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A Sal. A 75445 CENTRAL REFRIGERATION SYSTEM FOR A PROPOSED FOOD DISTRIBUTION CENTER IN NEW ORLEANS, LOUISIANA

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Food Distribution Research Laboratory
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Food Distribution Research Laboratory

FOREWORD

The Agricultural Research Service is making a study of food marketing facilities and methods in New Orleans, La. The purpose of this study is to determine the adequacy of existing facilities and handling methods; the extent to which food distribution costs can be reduced; the number of distributors who need new facilities; the kinds and sizes of facilities that are needed to correct existing defects; the cost of these facilities; their possible location; and the financial and other benefits that could be expected from developing a modern wholesale food distribution center for distributors who need to relocate.

Since most foods are perishable, adequate refrigeration is an essential feature of any food distribution center. To determine the nature and cost of such a refrigeration system for the facilities being planned and to hasten the completion of the study, Food Industry Services, a private contractor, was employed to do this phase of the work. This publication consists of the contractor's report exactly as it was written. The report is being published primarily for the use of the food wholesalers and others who may be involved in building the food distribution facilities in New Orleans, La.

Some of the material contained in this publication will be incorporated in the report ARS is publishing, which sets forth its total findings in the study of food marketing facilities and methods in New Orleans, along with its recommendations for their improvement.

K. H. Brasfield, Chief

Food Distribution Research Laboratory Agricultural Marketing Research Institute Agricultural Research Service

U. S. Department of Agriculture

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units. The same system would be used for heating the offices, except that a hot gas instead of a refrigerant would be circulated through the heat exchangers. The hot gas, furnished by the central plant, would also be used for defrosting evaporator coils in rooms with temperatures from -10° F to $+36^{\circ}$ F.

The original cost of a central refrigeration system for the New Orleans Food Distribution Center is estimated to be \$2.6 million. The cost of owning and operating the central refrigeration system is estimated at \$785,600 per year.

Charges to firms that use refrigeration from the central plant would be determined by assessing a flat charge for each terminal evaporator and by metering the demand for refrigerant liquids to each room. Different rates would be established for each size of evaporator and each suction temperature, and these would be based on the total annual cost for owning and operating the system.

REFRIGERATION REQUIREMENTS

The proposed New Orleans Food Distribution Center has 132 rooms that require 1,383 tons of refrigeration as shown in table 1.

Table 1.--Summary of refrigeration requirements by types of rooms and use

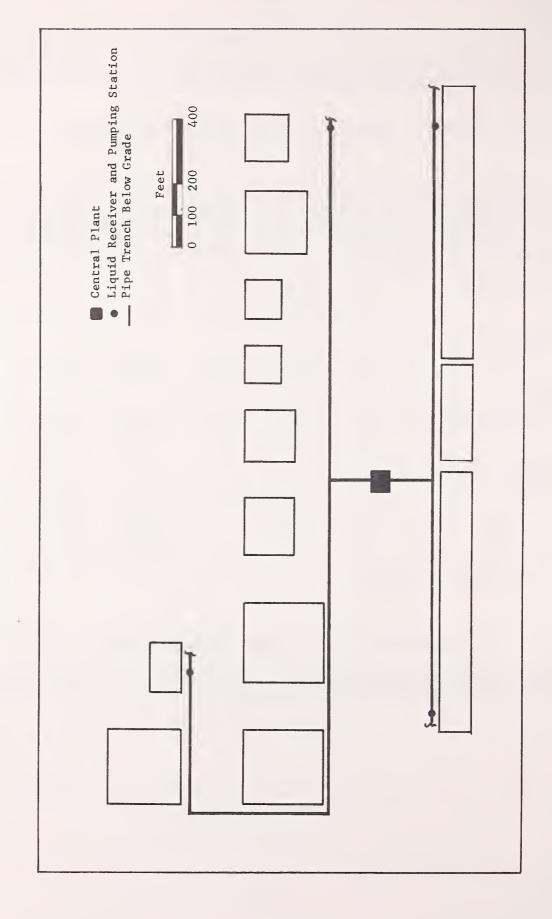
Types of rooms and temperatures	No. of rooms	No. of sq. ft	Ceiling heights (ft)	Tons of ref.	Sq. ft per ton
Freezer -10° to 0° F	12	12,232	12	46.5	263.1
	8	55,700	20	196.1	284.0
Cooler	38	70,775	12	324.7	218.0
+27° to +45° F	19	56,555	20	217.5	260.0
Work	3	15,600	12	85.7	182.0
+50° to +65° F	12	34,930	20	176.4	198.0
Office Nominal +72° F	$\frac{1}{40}$	100,105	8	335.9	298.0
Total	132	345,897		1,382.8	250.1

¹/ Groups of offices.

Description of the Proposed Central System

Figure 1 illustrates the layout of a central refrigeration system for the proposed New Orleans Food Distribution Center.

LAYOUT OF CENTRAL REFRIGERATION SYSTEM FOR THE PROPOSED FOOD DISTRIBUTION CENTER IN NEW ORLEANS, LA.



Refrigeration produced in the central plant would be confined to three suction temperatures, 1/ the minimum number required to serve each of the three temperature ranges required by firms in the food distribution center. The table below summarizes the suction temperature details.

Table 2.--Summary of suction temperatures and uses

Suction temperature use	Temperature range (in rooms)	Nominal suction temperature		
	(III I comb)	At terminal	At ref. plant	
Frozen product storage	-10° to 0° F	-30° F	-35° F	
Coolers	+32° to +45° F	+20° F	+15° F	
Workrooms and offices	+50° to +75° F	+35° F	+30° F	

The central refrigeration plant would also circulate hot refrigerant gas throughout the system for defrosting evaporator coils and heating offices.

Each pipeline trunk would consist of a maximum of three refrigerant circuits, one for each of the three suction temperatures, plus a hot-gas circuit and an evacuation line. Each refrigerant and hot-gas circuit would consist of two lines, one for transporting the liquid refrigerant or gas to the terminal evaporators, and the other for returning the gas and residue liquid to the plant for recompression, extraction of heat, and recirculation. One evacuation line would be installed in each quadrant to evacuate refrigerants from the system in the event of a breakdown of equipment or malfunction in the distribution lines.

Each building in the food distribution center would be served by branch circuits that emanate from the trunk lines and extend to the various rooms requiring refrigeration. Each room would be served by one or more evaporator units, sized for the particular requirements of the area, and connected to the appropriate refrigerant circuit. Although refrigerants would be delivered to terminal evaporators in coolers and freezers at one of the three suction temperatures (-30°, +20° or +35° F), the desired temperature range for a refrigerated room would be obtained by selection of the proper size of evaporator coils, installation of a back-pressure valve at each evaporator unit, and by simple adjustment of a thermostat installed in each room. To aircondition offices, a +35° F refrigerant would be circulated through heat exchangers to chill tapwater that, in turn, would be circulated through coils in

¹/ "Suction temperature" is defined as the temperature at which a refrigerant will vaporize at a specific pressure.

air-conditioning units. This same system can be used for heating as well as cooling by simply connecting the hot-gas circuits to the heat exchangers. Either hot or cold water would then be discharged from the heat exchangers and circulated through the air-conditioning units. The proper selection can be made by turning a control lever to the desired "summer" or "winter" position.

Required humidity levels could be obtained in all rooms by designing into the system the proper differential between the temperature of the refrigerant in the coils and the air temperature in the room. In instances where very high humidities are required (such as in wet coolers), a mechanical humidifier can be installed to disperse additional moisture in the air.

Facilities and Equipment

Central Plant Facilities

A building of approximately 9,800 square feet is required for the central refrigeration plant. In addition, an outside enclosure of 6,000 square feet is required for condensing equipment, water-cooling tanks, and header lines. Another 10,000 square feet of land is required for future expansion. A proposed location of the central plant is shown in figure 1, and a suggested layout of the proposed central plant facility is shown in figure 2. To accommodate high, vertical vessels and aid in ventilating the plant, the main part of the building should be constructed with a ceiling approximately 20 feet high. The areas of the plant to be used for support services (shops and offices) can be constructed with a lower ceiling of perhaps 12 feet. A cross section and a perspective of a typical refrigeration plant are shown in figures 3 and 4.

The proposed building can be erected at ground level with its walls constructed of masonry, concrete, or metal. Floor drains should be provided at appropriate locations. Floors should be sloped toward drain outlets 1/4 inch to the foot, constructed of high-density concrete, and painted to prevent spilled liquids from eroding the finish. Isolated concrete pads should be provided for the compressors and other major equipment. The building should be equipped with exhaust fans and air-intake vents to provide six air changes per hour. Lighting intensity should be provided at 75 foot-candles, tested 4 feet above the floor near critical equipment. Structural steel beams should be built into the roof over operating equipment to support at any one point as much as 3 tons of lift by an electric hoist.

Central Plant Equipment

To arrive at an estimate of the equipment required for a central refrigeration plant, it was necessary to decide first on the type of refrigerant best suited to the proposed system and then to select the equipment accordingly. After several alternatives had been evaluated, ammonia was chosen as the refrigerant. The primary reason for this choice is the greater refrigeration effect than other commonly used refrigerants. Smaller amounts are easier to

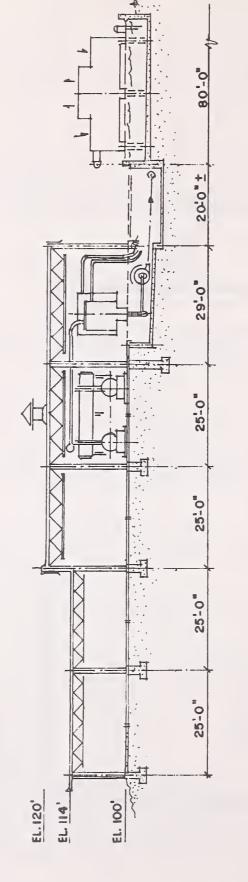
1/ Includes space for expansion of condenser area.

80'-0"(a) Condensers Distribu-**★** 20'-0'**¥** Headers tion Grade Below Accumulator-Intercoolers Pumps Reservoirs Suction and Low Stage High, | Intermediate (H.P. Receiver |Comp's Elect, 130'-0" Office & Toilets Control Room Shops 0 5 10 15 20 Feet 60'-0"

SUGGESTED LAYOUT OF PROPOSED CENTRAL REFRIGERATION PLANT.

FIGURE 2

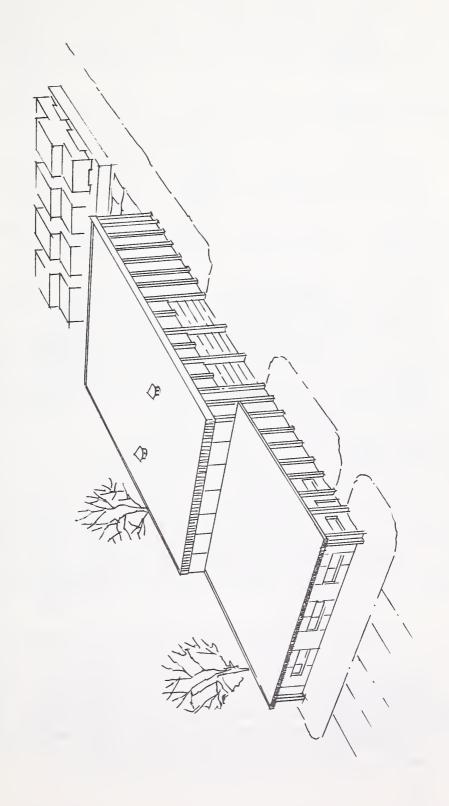
(a) Includes space for expansion of condenser area



1/ Includes space for expansion of condensers.

FEET

FIGURE 4
PERSPECTIVE OF PROPOSED CENTRAL REFRIGERATION PLANT.



circulate over the distances and in the quantities required for the complex. Ammonia can also be handled to supply the diversified range of temperatures required by the firms proposing to locate in the distribution center. A circulated, rather than an expanded, ammonia system was selected, because it is the only workable arrangement in view of the considerable piping pressure losses encountered and the variation in evaporation pressure required.

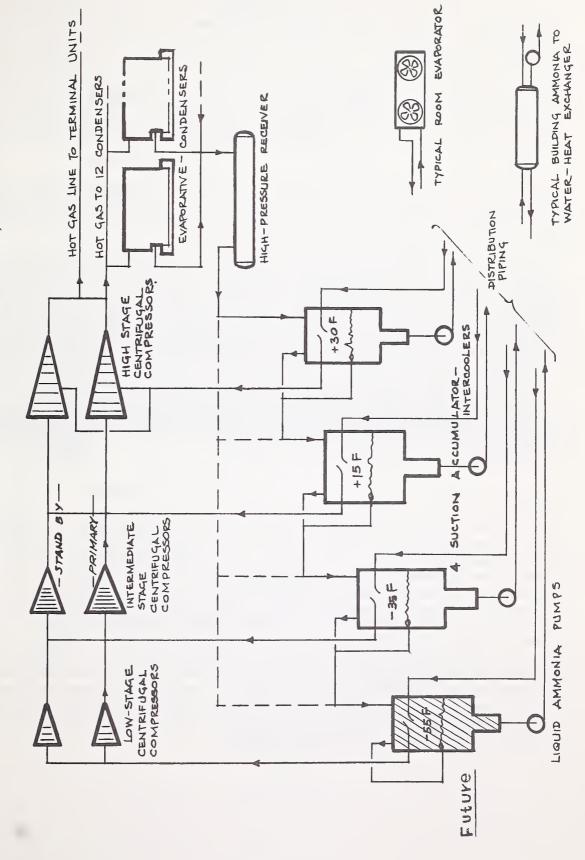
The primary equipment required in the central plant for compressing and circulating ammonia is shown in figure 5. The ammonia refrigerant would be circulated throughout the system, as follows:

- Ammonia liquid would be pumped from the bottom of each accumulator tank through a header and into the quadrant circuits to be transported to terminal evaporators.
- Ammonia gas (and liquid) returning from terminal evaporators would be circulated back into each accumulator.
- Ammonia gas from the top of each accumulator would be drawn off by the compressors and compressed into a hot gas.
- The hot ammonia gas would be circulated through the condensers, where the heat would be removed and the gas liquified.
- Ammonia liquid from the condensers would be stored in liquid receivers, then transferred back to the accumulators where the cycle would be repeated.

At the heart of the circulated-ammonia system would be large, centrifugal, multistage compressors, installed to operate in series and in parallel. Each group of compressors would be connected in series to operate as a "system". Each compressor in one group would have an identical counterpart in the other group, and the identical units would be connected in parallel so that they could operate interchangeably. In this way, if one compressor requires maintenance, another compressor can carry the load. On an average day, one group of compressors could sustain the refrigeration load of the system. On peak days, both groups of compressors would need to operate at only 80-percent capacity to produce the maximum of 1,383 tons of refrigeration required by the system.

Other equipment in the plant, such as condensers, accumulators, receivers, pumps, etc., would be selected to be compatible with the compressors. According to the plan outlined in this report, the central refrigeration system would require three accumulator tanks (one of each suction temperature) and six condensers. The proposed system would also require two large vessels to be used as liquid receivers, one tank for gas pump-out, one large refrigerant reserve storage tank, approximately 10 pumps for circulating liquid refrigerants, a detailed control panel, and an assortment or other support equipment such as motors, valves, and interconnecting pipelines.

SIMPLIFIED ARRANGEMENT OF AN AMMONIA-COMPRESSING AND CIRCULATING SYSTEM FOR A PROPOSED FOOD DISTRIBUTION CENTER FOR NEW ORLEANS, LA.



Approximately 75,860 lineal feet of pipelines in 5,300 lineal feet of trenches are required to circulate refrigerants to the 15 buildings in the food distribution center. A list of pipeline requirements and costs is shown in table 6. It is suggested that circulation lines be installed by burying them in the ground near the streets (see fig. 6). According to this plan, the refrigeration lines would be installed in a group and supported away from the sides and bottom of the trenches so that they could be easily insulated with foamed-in-place urethane. If installed in the manner illustrated in figure 6, all lines would be accessible for any necessary repair.

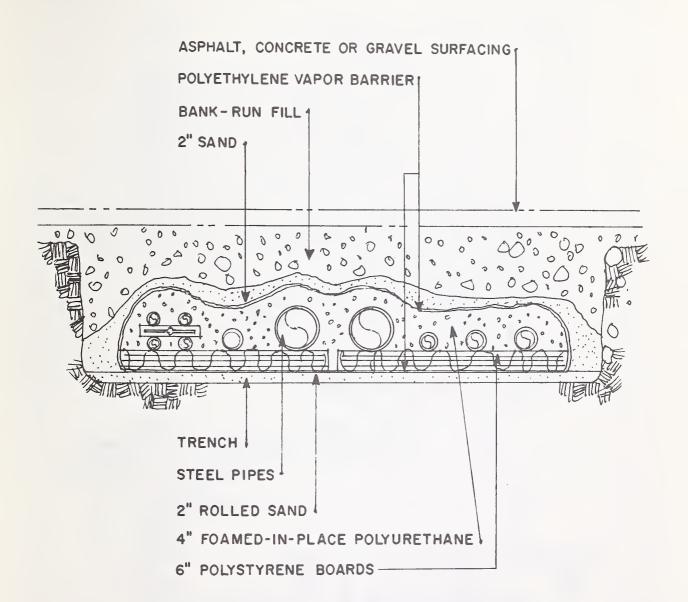
To facilitate the return of residual liquid to the central plant, it would be necessary to install at least one liquid receiver and accompanying pump in each distribution section, as shown in figure 1. Distribution lines would be sloped toward the receivers at an incline of 1 foot in 500 feet.

Final plans for installing the refrigerant distribution lines may depend on plans developed for installing other utility services in the area. As an alternative, the refrigeration lines could be installed in a walk-through tunnel, if such a facility were constructed for other utility services in the area. However, such an arrangement would significantly increase the cost of insulating the lines, since it would be necessary to install rigid insulation on the pipes manually rather than mechanically applied foamed-in-place urethane.

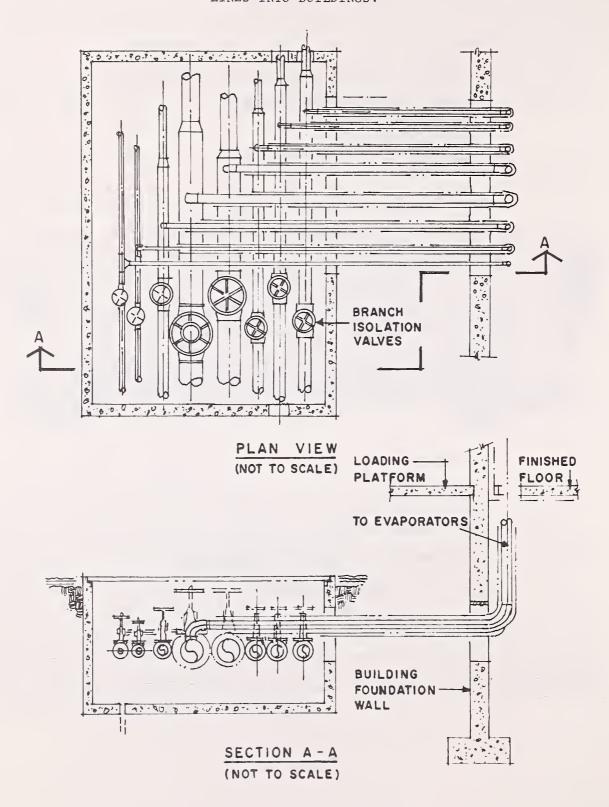
Control valves would be installed at points where branch lines run off the main circuits to service each building. Figure 7 illustrates a suggested method of extending branch lines into a typical building and distributing the refrigerants to the various terminal units.

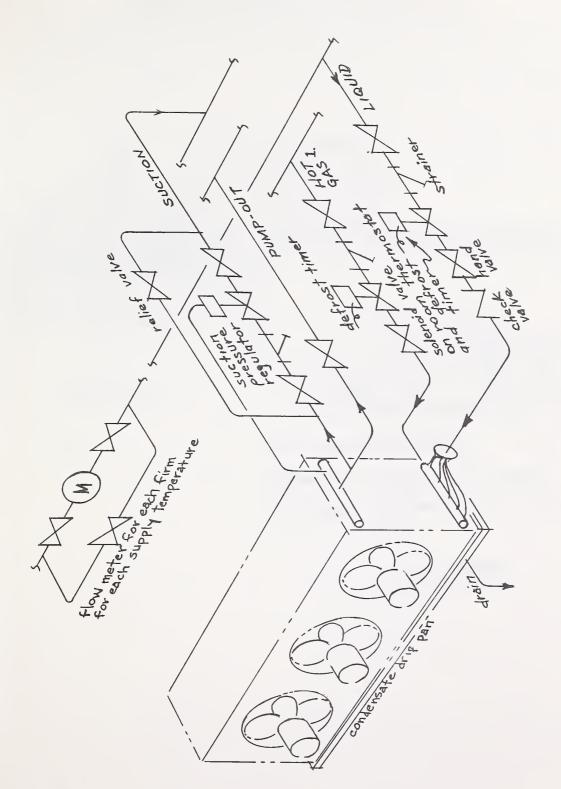
Terminal Equipment

Each room requiring refrigeration or air conditioning would be equipped with an evaporator unit consisting of coils, fans, and controls. These units would be installed as shown in figures 8 and 9. A summary of evaporator equipment and estimated costs appears in table 7. Each evaporator unit would be connected to one of three refrigeration circuits, according to the suction temperature required. Evaporator units that require defrosting would be connected to a hot-gas circuit. Each evaporator would be connected to an evacuation line and equipped with drip pans and drain lines leading to stub-up drains at the floor for collecting and disposing condensate. Evaporators used to air-condition offices would operate in conjunction with heat exchangers as explained in the previous section. Each unit would be equipped for regulating fresh air intake and the flow of air over the coils. One heat exchanger would be installed in each building and connected to the 35° F suction circuit, the hot-gas circuit, and the city water supply.



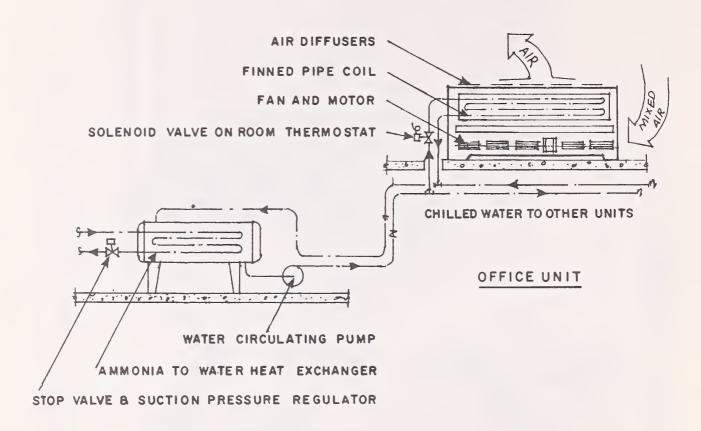
SUGGESTED METHOD OF EXTENDING BRANCH REFRIGERANT LINES INTO BUILDINGS.





1 Hot gas defrost lines would be omitted on units in rooms requiring above 36° F. temperatures of

FIGURE 9 TYPICAL COIL AND FAN UNIT AND HEAT EXCHANGER FOR AIR-CONDITIONING AND HEATING OFFICES.



BUILDING UNIT

Provisions for Expansion

Since it is anticipated that the peak of 1,383 tons of refrigeration for the food distribution center would occur less than 5 percent of the time, the system described in this report has a built-in excess capacity. This excess is estimated at 20 percent since the normal high demand is actually closer to 1,000 tons of refrigeration than it is to the 1,383 tons at peak requirements. For expansion beyond 20 percent, allowances are made in the central plant for a one-third increase in equipment without increasing the size of the building. Also, the central plant building can be extended to accommodate further expansion. To expand the refrigeration equipment capacity in the central plant, one or more complete groups of equipment can be added at a time. On this basis, the minimum addition would be three compressors accompanied by necessary equipment such as accumulators, receivers, etc.

The pipelines for the proposed refrigerant circuits are sized to accommodate a 20-percent increase in the capacity of each suction temperature. Additional expansion can be obtained by installing more circuits adjacent to the initial lines (see fig. 6).

COST OF FACILITIES AND EQUIPMENT

The total cost of a central refrigeration system is estimated to be a maximum of \$2.6 million. A breakdown of this cost is summarized in table 3 below and further detailed in tables 4 through 7. The estimated cost of owning and operating the central refrigeration system, including the terminal units, is estimated at \$785,600, as shown in table 8.

Table 3.--Summary of estimated costs of facilities and equipment for the proposed central refrigeration system

Facilities and equipment $\frac{1}{2}$	Reference	Estimated cost
Central plant facility Central plant equipment Distribution lines Terminal equipment Cost per ton	Table 4 Table 5 Table 6 Table 7 <u>2</u> /	\$172,500 864,800 644,300 880,800 (1,853)
Total		2,562,400

 $[\]underline{1}$ / Land is excluded, because this item is part of the total cost of the food distribution center.

Table 4.--Estimated costs of central refrigeration plant

Item	Costs
Building shell 9,800 ft 2 @ \$10/ft 2	\$98,000 30,000 4,000 8,000 10,000 22,500
Total	172,500

 $[\]underline{1}/$ A liberal allowance of 15 percent was used throughout to assist in establishing maximum estimated costs.

^{2/} \$2,562,400 divided by 1,382.8 tons = 1,853 (not included in total).

Table 5.--Summary of estimated costs of equipment for central refrigeration plant $\underline{1}/$

Description of equipment	Estimated cost to contractor
Two centrifugal compressors	\$178,000
Drive motors, high-speed gear units, related items	64,000
Condensers and related items	77,000
Receivers, intercoolers, tanks, auxiliary compressors, and related items	63,000
Special valves, level controls, sensors, and fittings	48,000
Fabricated piping and insulation	58,000
Integrated automatic controls	73,000
Electrical transformers	13,000
Installation $\underline{2}/$	178,000
Fees and contingencies (15%)	112,800
Tota1	864,800

 $[\]frac{1}{2}/$ Based on estimates from manufacturers and contractors. $\frac{2}{2}/$ Includes \$65,000 for refrigerant and includes markup on equipment and materials.

Table 6.--Estimated costs of refrigerant distribution lines $\underline{1}/$

Diameter of pipe	Lengths required	Pipe cost per 100 feet	Cost <u>2</u> /
Inches	Lineal feet		
14	340	\$665	\$2,300
12	220	525	1,200
10	3,000	410	12,300
8	4,800	295	14,200
6	3,800	205	7,800
5	6,100	150	9,200
4	6,000	120	7,200
	3,800	105	4,000
3	7,200	90	6,500
$2\frac{1}{3}$	6,300	65	4,100
2	5,400	40	2,200
$3\frac{1}{2}$ 3 $2\frac{1}{2}$ 2 $1\frac{1}{2}$	9,300	30	2,800
1	15,600	25	3,900
<u>1</u> 2	4,000	25	1,000
Subtotal	75,860		78,700
Receivers, pumps,	62,500		
Excavation, backfi	73,100		
Installation of pi	222,600		
Insulation install	123,400		
Fees and contingen	84,000		
Total	644,300		

^{1/} Based on estimates from manufacturers and contractors. 2/ Figures in Columns 2 and 4 rounded to nearest 100. Column 4 = Column 2 x Column 3.

Table 7.--Estimated costs of terminal refrigeration equipment $\underline{1}/$

Type of room	Room temperature range <u>2</u> /	No. of rooms	No. of sq. ft	No. of sizes	No. of units	Cost to contractor 2/
Frozen product	-10° to 0° F	20	67,932	9	37	99\$
Coolers low temperature	+27° to 45° F	57	127,330	75	123	184,500
Workrooms	+50° to 65° F	15	50,530	ĸ	67	83,300
Office air conditioning	+65° to 75° F	40	100,105	7	163	81,500
Subtotal		132	345,897	16	372	415,900
Heat exchangers $3/$				12	27	43,700
Controls, insulation, etc. $\frac{4}{4}$	· . 4/			1	1	118,500
Installation $5/$				ı	ı	187,800
Fees and contingencies (15%)				1	1	114,900
Total				1	1	880,800

Valves, thermostats, hardware, pipe, and sundry supplies, including electrical accessories. 1/ Based on estimates from manufacturers and contractors. $\frac{2}{2}$ / Figures in Column 7 rounded to nearest \$1,000. $\frac{3}{4}$ / For air conditioning. $\frac{4}{4}$ / Valves, thermostats, hardware, pipe, and sundry supplies $\frac{4}{5}$ / Includes markup on equipment and materials.

Table 8.--Estimated annual cost of owning and operating the proposed central refrigeration system $\underline{\mathbf{1}}/$

	Footnote	Annua1	
Item	reference	costs	Tota1
	number	(rounded)	
Building occupancy		46.000	
Depreciation @ 25 years	1	\$6,900	
Maintenance and repairs	2	3,500	AAF AAA
Cost of financing	3	15,500	\$25,900
Central plant equipment			
Depreciation	4	57,700	
Parts and supplies	5	8,700	
Refrigerant	6	3,300	
Electrical power	7	129,500	
Miscellaneous	8	5,700	
Cost of financing	9	77,800	282,700
Distribution lines		10.000	
Depreciation	10	43,000	
Parts and supplies	11	6,500	
Cost of financing	12	58,000	107,500
Terminal equipment			
Depreciation	13	58,700	
Parts and supplies	14	8,900	
Cost of financing	15	79,300	146,900
Administration of system			
Administration of system Payroll	Table 9	132,000	
·	16	13,200	
Other expenses			151 200
Vehicle lease and operation-	17	6,000	151,200
0.1 1			71/ 000
Subtotal			714,200
Earnings and reserve minus 10%			71,400
Total	18		785,600
			Í
Deduct terminal equipment			146,900
Net total for calculating	1.0		(00 700
refrigeration charges	19		638,700

/ See footnotes to table 8, page 23, for explanations for Column 3.

- 1. Cost of central plant building: \$172,500 from table 4 divided by 25 years' depreciation = \$6,900.
- 2. 1/2 of depreciation rate: \$6,900 divided by 2 = \$3,450.
- 3. 9-percent average annual interest and other financing charges: $$172,500 \times 0.09 = $15,500 \text{ average first 3 years.}$
- 4. Cost of central plant equipment \$864,800 from table 5 divided by 15 years' depreciation = \$57,700.
- 5. Parts and supplies = 1 percent of cost, i.e., $$864,800 \times 0.01 = $8,648$.
- 6. Refrigerant replacement cost = 5 percent of \$65,000 = 3,250 initial cost. See table 5, footnote 3.
- 7. \$129,500. Based on a calculated 14 million kWh at 80 percent of capacity and calculated at New Orleans industrial rate x 15 percent for contingencies = \$0.00925.
- 8. 10 percent of depreciation: $$57,000 \times 0.10 = $5,700$.
- 9. 9-percent average annual interest and other financing charges: $$864,800 \times 0.09 = $77,800$ average first 3 years.
- 10. Cost of distribution lines: \$644,300 from table 6 divided by 15 years' depreciation = \$43,000.
- 11. Parts and supplies = 1 percent of cost, i.e., $$644,300 \times 0.01 = $6,443$.
- 12. 9-percent average annual interest and other financing charges: $$644,300 \times 0.09 = $58,000 \text{ average first 3 years.}$
- 13. Cost of terminal equipment: \$880,800 from table 7 divided by 15 years' depreciation = \$58,700.
- 14. Parts and supplies = 1 percent of cost, i.e., $$880,800 \times 0.01 = $8,808$.
- 15. 9-percent average annual interest and other financing charges: $\$880,800 \times 0.09 = \$79,300$ average first 3 years.
- 16. 10 percent of payrol1: $$132,000 \times 0.10 = $13,200.$
- 17. 30,000 miles @ 20 cents per mile for one truck = \$6,000.
- 18. \$785,600 divided by 1,382.8 tons of refrigeration = \$568 per ton.
- 19. \$638,700 divided by 1,382.8 tons of refrigeration = \$462 per ton.

Table 9.--Estimated personnel required for maintenance, repairs, and administration of the proposed central refrigeration system

Assignment	Number of people	Annual cost
Manager and chief engineer	1	\$15,000
lst-class operating engineers	1	11,000
2nd-class operating engineers	1	10,000
Chief electrician	1	10,000
Chief pipefitter	1	10,000
Electricians	2	16,000
Pipefitters	2	16,000
Welders	1	8,000
Meter reader	1	8,000
Supply clerk	1	8,000
Administrative clerk	1	8,000
Subtotal	13	120,000
10-percent payroll burden		12,000
Total		132,000



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